Eco-Vehicle Speed Control at Signalized Intersections using I2V Communication

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Overview

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Introduction

 The research develops an eco-speed control system to reduce vehicle fuel consumption in the vicinity of signalized intersections.



Similar Research

Author	Findings	Shortcomings
Barth et al. [3]	 Studied TSS to drivers using CMS and in-vehicle devices. Found 40% savings 	• Used TTR info to advise drivers not to slow down if red is near.
Asadi & Vahidi [4]	 Developed a cruise control which reduces Pr(reach stop-bar @ red). Showed 47% savings. 	• Alternate speed profiles not studied using fuel consumption models.
Tielert et al. [5]	• Used VISSIM simulation to find factors affecting fuel savings if I2V communication is present	• Used PHEM model for comparison and not optimization.
Malakorn & Park [6]	 Studied a CACC based on I2V min{length of dec & acc} & min{idling time} 	No FC model in objective.Downstream neglected.
Mandava et al. [7]	 Optimal instantaneous velocity to drivers using TSS. min{rate of dec/acc} 	 No FC model in objective

Model Description

- Previous publications used a simplified objective function.
- Here, the system computes a "proposed time to reach intersection" using
 - SPaT information
 - Queued vehicle information
 - Approaching vehicle information
- Computes a "proposed fuel-optimal trajectory" using
 - Vehicle deceleration and acceleration models
 - Microscopic fuel consumption models
 - Roadway characteristics

Model Description



Model Logic

- Signal is currently GREEN
 - Case I: GREEN will continue so that vehicle can pass through at current speed.
 - Case 2: GREEN will end soon but vehicle can legally pass through intersection during the green or yellow indication if it speeds up within speed limit.
 - Case 3: GREEN will end soon and vehicle cannot pass during this phase.
- Signal is currently RED
 - Case 4: RED will continue but vehicle needs to be delayed to receive GREEN indication.
 - Case 5: RED will end soon so that vehicle will receive GREEN when it reaches stop-line at current speed.

Model Logic

- Cases 1,2, 3 and 5 are fairly simple
- Case 4 requires trajectory optimization every time step within detection zone.
- Min{fuel consumed}
- Subject to
 - Fixed travel distance upstream.
 - Fixed time to reach intersection.
 - Variable speed at intersection.
 - Vehicle acceleration characteristics downstream to accelerate back to initial speed.



Model Logic

- Speed trajectory at intersection is divided into:
 - Upstream section (deceleration to achieve delay) &
 - Downstream section (accelerate to original speed)
 - Cruising section to maintain a constant distance of travel.



Deceleration Model

TTG = t seconds DTI = x meters Approach speed = v_a m/s Speed at signal = v_s m/s Delay required = Δt seconds Veh. deceleration = d m/s²

Cruising dist. = x_r m



Conserve x and t:

$$x = \frac{v_a^2 - v_s^2}{2d} + x_r$$
 and $t = \frac{v_a - v_s}{d} + \frac{x_r}{v_s}$

Combining them: $t = \frac{v_a - v_s}{d} + \frac{1}{v_s} \left(x - \frac{v_a^2 - v_s^2}{2d} \right)$

Solving for v_a :

$$v_s = v_a - d \cdot t + \sqrt{d\left(d \cdot t^2 - 2v_a t + 2x\right)}$$

For any v_a , x_r is given by: $x_r = x - \frac{v_a^2 - v_s^2}{2d}$

Acceleration Model

- Rakha & Lucic Model [8] was used.
 - Vehicle dynamics model.
 - Acceleration = Resultant Force/mass
 - Resultant Force = Tractive Force Resistive Force

$$F = \min\left(3600 f_p \beta \eta_d \frac{P}{v}, m_{ta} g \mu\right)$$

$$R = \frac{\rho}{25.92} C_d C_h A_f v^2 + mg \frac{c_{r0}}{1000} (c_{r1}v + c_{r2}) + mgG$$

Fuel Consumption Model

- Virginia Tech Comprehensive Power-based
 Fuel Model (VT-CPFM) Type 1²¹.
 - Based on instantaneous power

$$FC(t) = \frac{\alpha_0 + \alpha_1 P(t) + \alpha_2 P(t)^2}{\alpha_0} \quad \forall P(t) \ge 0$$
$$\forall P(t) < 0$$

- Parameters α_0 , α_1 and α_2 can be calibrated using EPA fuel economy ratings.
- Does not result in a bang-bang control
 - Optimum acceleration is not necessarily full throttle acceleration

Example Illustration

- Simulation was conducted for different approach speeds considering the following parameters:
 - TTG = t = 14 s
 - DTI = x = 200 m
 - Approach speed = v_a = 20 m/s
 - Delay required = $\Delta t = 4$ s
 - $d_{min} = 0.82 \text{ m/s}^2$ (computed)
 - $d_{max} = 5.90 \text{ m/s}^2$ (limiting).

Example Illustration



Driving Transportation with Technology



Simulation Results



Acceleration Fuel (I)

Upstream Fuel (I)

Fuel consumed in seven cases of 30% throttle by Chevy Malibu (I)



Case Studies

- Experiment repeated using various sets of
 - Approach speeds
 - Desired delay estimates
 - Vehicle Types
- 80 cases simulated maintaining a constant DTI of 200 m.



 $FC_i(ds) = FC_i(v_s \rightarrow v_a) + FC_{cruise}(v_a) \times [x_{\max} - x_{i-acc}]$

Case Studies

- Four vehicles were tested:
 - Vehicles selected were available at VTTI and thus were validated using field measurements

	Vehicle I	Vehicle 2	Vehicle 3	Vehicle 4
Vehicle Info	SAAB	Mercedes	Chevy	Chevy
Model	95	R350	Tahoe	Malibu
Year	2001	2006	2008	2007
Engine Size (L)	2.3	3.5	5.3	2.2
EPA Rating (City/Highway)	21/30	6/21	14/20	24/34
Fuel-optimal speed	45.9mph	37.3mph	37.3mph	41.6mph

Sample Results (Fuel-consumption matrix)



Fuel Consumption Matrix (when $v_a = 45$ mph and $\delta t = 4$ seconds)

Inference I: The greater the acceleration level, the higher is the fuel consumed.

Sample Results (fuel consumed in ml at 20% throttle)

 Results from two separate simulated cases are shown below (for 20% throttle) and are color coded according to fuel consumed.

Va =	Va = 20m/s,TTG = 14s, DTI = 200m			00m	Va =	l I m/s,TT	G = 22s	, DTI = 2	00m
dec(m/s ²)	SAAB	R350	TAHOE	MALIBU	dec(m/s ²)	SAAB	R350	TAHOE	MALIBU
0.8163	50.90	76.00	59.20	44.60	0.1736	20.20	23.90	27.90	17.90
I	47.50	70.00	55.50	42.20	0.25	20.10	23.90	27.30	18.00
I.25	47.00	67.00	53.00	41.70	0.5	20.30	24.20	27.40	18.60
I.5	46.00	67.50	52.40	42.20	0.75	21.00	24.20	27.20	18.90
I.75	45.70	66.90	53.20	42.00	I	21.20	24.50	27.30	18.80
2	45.40	66.40	52.80	41.60	١.5	21.20	24.50	27.30	18.80
2.5	45.10	65.90	52.20	41.40	2	21.40	24.80	27.40	18.90
3	46.00	65.40	51.80	41.20	3	21.40	24.80	27.50	18.90
4	45.70	65.40	51.70	41.10	4	21.40	24.80	27.50	19.00
5	45.70	64.90	51.30	41.90	5	21.40	24.80	27.50	19.00
	•		• 1					•	

Inference 2: Fuel-optimal case may not always involve minimal deceleration level

Sample Results (deceleration in m/s² in optimum case)

Chevy Tahoe						Che	vy Ma	libu			
Approach Speed (mph)						_	Арр	roach S	peed (r	nph)	
	_	25	35	45	55			25	35	45	55
	2	1.00	2.00	1.00	4.75		2	0.25	0.50	I.75	2.50
(s)	4	5.75	3.50	5.75	5.00	(s)	4	5.75	I.25	5.75	3.00
lay	6	2.75	5.00	5.75	5.50	lay	6	0.25	I.00	5.75	5.50
De	8	3.25	5.75	4.50	5.75	De	8	0.75	5.75	5.75	5.50
	10	3.75	5.75	5.25	5.75		10	1.00	5.75	4.75	4.25

Inference 3:

Deceleration in fuel-optimal case is proportional to

(a) Approach Speed(b) Delay to be induced in the trajectory

Sample Results

(% difference between worst case and best case)

		S	AAB 9	5				
	_	Аррі	roach Sp	beed (r	nph)			
	_	25	25 35 45 55					
	2	11%	70%	91%	104%			
(s)	4	27%	54%	81%	86%			
lay	6	21%	43%	66%	71%			
De	8	20%	43%	56%	60%			
	10	20%	35%	51%	59%			
			– 1					

		Ch	evy Tah	ioe	
	_	Арр	roach S	peed (n	nph)
		25	35	45	55
	2	21%	102%	134%	154%
(s)	4	38%	79%	117%	130%
lay	6	30%	55%	102%	110%
De	8	28%	64%	89%	96%
	10	27%	54%	81%	98%

		Che	evy Ma	libu					
	Approach Speed (mph)								
	_	25 35 45 55							
	2	10%	67%	88%	96%				
(s)	4	27%	54%	76%	85%				
lay	6	20%	40%	64%	71%				
De	8	19%	42%	57%	62%				
	10	22%	35%	52%	63%				

Mercedes R350									
	_	Approach Speed (mph)							
		25	25 35 45 55						
	2	19%	90%	110%	118%				
(s)	4	38%	70%	93%	98%				
lay	6	30%	53%	78%	83%				
De	8	28%	53%	67%	68%				
	10	29%	45%	62%	71%				

MATLAB Application

X scospeedmodule Eco-Speed Control Application - Version 2.0 Vehicle Selection Saab '95 Mercedes R350 Select Select an available vehicle: Chevrolet Tahoe Chevrolet Malibu Or Define Your Vehicle: Details 2-Details 1 Altitude Correction Factor (Ch): 0.95 Vehicle Name: My vehicle Rolling Resistance Constant 1: Vehicle Power (kW): 1.75 138 Vehicle Mass (kg): Rolling Resistance Constant 2: 1600 0.0328 Rolling Resistance Constant 3: Vehicle Frontal Area (m²): 2.29 4.575 Road Grade (%): Drag Coefficient (Cd): 0.0 0.29 Driveline Efficiency (0~1): VT-CPFM 1 Parameters: 0.92 % Mass on Tractive Axle (0~1): Alpha 0: 0.00050809 0.54 Alpha 1: Coefficient of Friction: 0.000091079 0.8 Density of Air (kg/m^3): 1.2256 Alpha 2: 0.000001 << BACK NEXT >>

MATLAB Application

ecospeedmodule

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Eco-Speed Control Application - Version 2.0

-Scenario Description-

Enter intersection and arterial characteristics:

	Passive Data	
200	Length of Intersection Queue (m):	inactive
20	Safety Interval (s):	inactive
14	Arterial Jam Density (veh/mile):	inactive
20	Saturation Flow Rate (veh/hr):	inactive
5	Avg. Vehicle Acceleration (m/s/s):	inactive
		NEXT >>
	200 20 14 20 5	200Length of Intersection Queue (m):20Safety Interval (s):14Arterial Jam Density (veh/mile):20Saturation Flow Rate (veh/hr):5Avg. Vehicle Acceleration (m/s/s):

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MATLAB Application

📣 ecospeedmodule

Eco-Speed Control Application - Version 2.0

Optimization Tool-

My vehicle is ready for Eco-Speed Control. Hit 'Optimize!' button to find the most fuel optimal path.

decel. (m/s/s) vel. at int (m/s) 0.2 0.3 0.4 0.5 0.6 0.7 0.8 0.9 1.0 1 0.8163 8.5714 0.0509 0.0559 0.0599 0.0635 0.0672 0.0711 0.0742 0.0744 0. 2 1 12,0000 0.0475 0.0514 0.0538 0.0566 0.0603 0.0631 0.0654 0.0678 0. 3 1.2500 12.8078 0.0466 0.0494 0.0526 0.0553 0.0583 0.0619 0.0630 0.0654 0 4 1.5000 13.1774 0.0460 0.0487 0.0522 0.0547 0.0573 0.0606 0.0617 0.0641 0. 5 1.7500 13.3955 0.0457 0.0485 0.0518 0.0540 0.0564 0.0593 0.0623 0.0624 0 6 2 13.5407 0.0454 0.0492 0.0510 0.0540 0.0564 0.0593 0.0604 0.0627 0. 7 2.2500 13.6446 0.0454 0.0490 0.0514 0.0544 0.0554 0.0597 0.0608 0.0632 0 8 2.5000 13.7228 0.0451 0.0490 0.0507 0.0534 0.0554 0.0597 0.0609 0.0634 0. 9 2.7500 13,7839 0.0534 0.0459 0.0485 0.0507 0.0555 0.0581 0.0609 0.0635 0 13.8329 0.0534 10 3 0.0460 0.0485 0.0507 0.0555 0.0581 0.0610 0.0611 0 11 3.5000 13.9069 0.0460 0.0483 0.0503 0.0538 0.0558 0.0584 0.0614 0.0616 0. 12 4 13.9600 0.0456 0.0484 0.0503 0.0528 0.0557 0.0584 0.0594 0.0617 0 13 4.5000 0.0503 0.0528 0.0594 14.0000 0.0457 0.0484 0.0557 0.0585 0.0618 0 14 5 14.0312 0.0457 0.0484 0.0503 0.0528 0.0557 0.0585 0.0595 0.0619 0 15 5.5000 14.0563 0.0453 0.0479 0.0503 0.0528 0.0557 0.0585 0.0595 0.0619 0. ٠. ш

The optimum case is decelerating at 2.5 m/s/s for 2.51087 seconds and past signal, accelerating at 20 % throttle. Click RESULTS to plot the optimum velocity trajectory.

<< BACK

RESULTS >>

Optimize!

Conclusions

- Presentation demonstrates that objective function
 - Should not be simplified
 - Need to include a fuel-consumption model
 - Model should be robust
 - Need to incorporate entire downstream and upstream maneuver.
- Fuel-optimum trajectory is case-specific and depends on many factors.
 - Does not necessarily imply minimum deceleration level
- Potential savings for approaching vehicle:
 - 53% for sedans and 65% & 80% for the R350 & Tahoe.

Conclusions

- Deceleration upstream is case-specific.
- Initial deceleration is proportional to approach speed.
- Initial deceleration is also proportional to required delay.
- Acceleration depends on
 - Speed at intersection
 - Function of deceleration level
- In-vehicle module demonstrated with MATLAB application.
- Accelerating at lowest throttle level
 - Most fuel-optimal downstream action, but reduces discharge rate.
- Possible fuel savings is proportional to engine-size and approach speeds.

References

- S. C. Davis, S.W. Diegel, and R. G. Boundy, *Transportation Energy Data Book*, vol. 91. Oak Ridge, TN: , 2010, p. 385.
- 2. A. Bandivadekar et al., On the road in 2035: Reducing transportation's petroleum consumption and GHG emissions, no. July. 2008, p. 196.
- G.Wu, K. Boriboonsomsin, W.-B. Zhang, M. Li, and M. Barth, Energy and Emission Benefit Comparison of Stationary and In-Vehicle Advanced Driving Alert Systems, Transportation Research Record: Journal of the Transportation Research Board, vol. 2189, no. 1, pp. 98-106, Dec. 2010.
- 4. B.Asadi and A.Vahidi, Predictive Cruise Control: Utilizing Upcoming Traffic Signal Information for Improving Fuel Economy and Reducing Trip Time, Control Systems Technology, IEEE Transactions, pp. 1-9, 2010.
- T. Tielert, M. Killat, H. Hartenstein, R. Luz, S. Hausberger, and T. Benz, The impact of traffic-light-to-vehicle communication on fuel consumption and emissions, in Internet of Things (IOT), 2010, 2010, pp. 1–8.
- K. J. Malakorn and B. Park, Assessment of mobility, energy, and environment impacts of IntelliDrive-based Cooperative Adaptive Cruise Control and Intelligent Traffic Signal control, in Sustainable Systems and Technology (ISSST), 2010 IEEE International Symposium, 2010, pp. 1–6.

References

- S. Mandava, K. Boriboonsomsin, and M. Barth, Arterial velocity planning based on traffic signal information under light traffic conditions, in Intelligent Transportation Systems, 2009. ITSC'09. 12th International IEEE Conference on Intelligent Transportation Systems., 2009, pp. 1–6.
- 8. H. Rakha, M. Snare, and F. Dion, *Vehicle dynamics model for estimating maximum lightduty vehicle acceleration levels*, Transportation Research Record: Journal of the Transportation Research Board, vol. 1883, no. 1, pp. 40–49, Jan. 2004.
- H.A. Rakha, K.Ahn, K. Moran, B. Saerens, and E.V. D. Bulck, Virginia Tech Comprehensive Power-Based Fuel Consumption Model: Model development and testing, Transportation Research Part D: Transport and Environment, Jun. 2011.





Go Hokies!

Thank You!